Amendments to the Specification:

**Page 7**, please amend the two consecutive paragraphs beginning on line 4 as follows:

The compressor includes a sealing arrangement between the rear-facing surface of the compressor wheel **30** and the fixed wall **20** of the compressor housing. The space between the wheel and the fixed wall represents a leakage pathway through which high-pressure air and gaseous fuel could leak into the bearing casing. At least part of the leaking pathway is defined between the rear surface of the compressor wheel **30** and the fixed wall **20** of the compressor housing. As previously noted, leakage of fuel into the bearing casing must be prevented. The sealing arrangement includes a hydraulic resistance element or seal **42** spaced radially outward of the bearing casing. Thus, a portion **44** of the leakage pathway is defined between the hydraulic resistance element **42** and the bearing casing. The portion **44** is free of any hydraulic resistance elements. The hydraulic resistance element **42** is shown as a labyrinth seal, although other types of seals for providing a high hydraulic resistance (e.g., brush seals) can be used instead. At least one pressurized air supply duct **46** extends through the compressor housing into the portion **44** of the leakage pathway.

In operation, pressurized air (as indicated by arrow **48**) is supplied via the supply duct **46** into the pathway portion **44**. The air is "clean", i.e., free of fuel. The pressurized air is supplied at a pressure exceeding that in the main gas flow path of the compressor (i.e., the pressure at the radially outward side of the outer seal **42**). Accordingly, a portion **50** of the pressurized air will flow inwardly along the pathway portion **44** into the bearing casing. This air is evacuated from the bearing casing, along with oil that has already lubricated the bearing, through a drain duct **52**. The air and oil can then be processed in an air-oil separator or the like, so that air that has been cleaned of oil vapors can be discharged to atmosphere, while the oil can be recovered. The remainder **54** of the pressurized air supplied to the pathway portion **44** will flow outwardly past the seal **42** into the main gas flow path of the compressor, and ultimately into the discharge duct **18**. Thus, the sealing arrangement effectively eliminates virtually all possibility of fuel leakage into the bearing casing, such that substantially no fuel escapes from the compressor. The relative

proportions of the air flows into the bearing area and out to the main gas flow path can be controlled by design procedures that are within the routine capability of those skilled in the art.

**Page 8**, please amend the paragraph beginning on line 8 as follows:

A second embodiment of the invention is shown in FIG. 2. The compressor **110** of FIG. 2 is substantially similar to the compressor **10** of FIG. 1 (having a leakage pathway at least part of which is defined between the rear surface of the compressor wheel **30** and the fixed wall **20** of the compressor housing), except as noted below. The sealing arrangement of the compressor **110** includes a set of auxiliary blades **60** mounted on the rear surface of the compressor wheel **30**, and may also include a hydraulic resistance element or seal **40** as shown. The auxiliary blades are spaced radially outward of the seal **40** such that a cavity or portion **44** of the leakage pathway is defined between the blades and the seal. A pressurized air supply duct **46** leads into the portion **44** of the leakage pathway. The auxiliary blades are configured to draw air radially outwardly through the blades and compress the air to a higher pressure. Accordingly, in this embodiment, the pressurized air **48** supplied through the supply duct **46** can be supplied at a pressure less than that in the main gas flow path of the compressor, but higher than the pressure in the bearing casing.

**Page 10**, please amendment the paragraph beginning on line 13 as follows:

FIG. 5 shows a compressor **210'** generally similar to that of FIG. 3 (having a leakage pathway at least part of which is defined between the rear surface of the compressor wheel **30** and the fixed wall **20** of the compressor housing), but having an air or magnetic (or combination air/magnetic) bearing **24'** instead of an oil-lubricated bearing. The innermost hydraulic resistance element is omitted in this embodiment, although it may be included if desired. Thus, the sealing arrangement comprises hydraulic resistance elements or seals **42** and **70** that are radially spaced apart to define a cavity **72** between them. The leakage pathway portion **44** that extends radially inward from the inner seal **42** into the bearing casing is free of any further hydraulic resistance elements. The pressurized air supply duct **46** leads into this pathway portion **44**. A recirculation duct **74** extends from the cavity **72** back to the compressor inlet duct **14**. In

operation, clean pressurized air **48** is fed into the pathway portion **44** at a pressure higher than that in the bearing casing **22** but lower than that in the main gas flow path of the compressor. One portion **50** of the air will flow inwardly along the pathway portion **44** into the bearing casing. The remainder of the air **76** will flow outwardly past the seal **42** into the cavity **72**. Since the pressure in the cavity **72** is lower than that in the main gas flow path, some air and gaseous fuel will flow from the main gas flow path inwardly past the outer seal **70** into the cavity **72**, as indicated by arrow **78**. The air and fuel in the cavity **72**, however, is still at a higher pressure that than in the inlet duct **14**, and hence this air and fuel will flow through the recirculation duct **74** back into the inlet duct **14**. In this manner, fuel is prevented from leaking into the bearing casing and therefore substantially no fuel escapes from the compressor.